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## **Discussion Paper**

### **Calculating and Comparing Swaybar Link Compliance.**

How to calculate relative stiffness of different types of links using compliant materials. Example used is for Subaru rear alloy links fitted with polyurethane bushes.

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Assuming we would like to compare the relative stiffness of different link materials where the links are otherwise identical except one has compliant bushes at both ends.

### **Non-compliant Links**

Assuming the link is just a straight tension-compression member, the stiffness of the non-compliant link is simply the stiffness of the material it's made of, which can be calculated as:

$$K (\text{non-comp}) = K$$

Where  $K = A \cdot E / L$   
A = cross sectional area  
E = material modulus  
L = length

### **Compliant Links**

For the compliant link we have in effect 3 springs in series, the 2 softer springs at either end representing the compliant bushes. The effective spring rate of this is:

$$K (\text{comp}) = \frac{k \cdot K}{2 \cdot K + k} \quad \text{up until the bushes max out in travel}$$
$$= K \quad \text{for travel beyond the bush max out travel.}$$

Where  $K$  = stiffness as above in non-compliant case  
 $k$  = stiffness of the individual bushes

This gives a low stiffness value up until the bushes reach their limit, then it has the same stiffness as the non-compliant link.

For example, assume the link is made of AL (aluminium) and is 20 mm x 15 mm x 100 mm long, this delivers a stiffness of 210 000 N/mm for a non-compliant.

The bushes each have a stiffness of 20 000 N/mm but can only travel a max of 2 mm, therefore the stiffness of the compliant link is only 9545 N/mm up to a deflection of 4 mm (2 bushes at a max of 2mm). From there on it will operate at 210 000 N/mm. See figure below.

### Forces VS Deflection example.

